

High-Fidelity Simulation of Unsteady Flow Problems using a 3rd Order Hybrid MUSCL/CD scheme

ECCOMAS, June 6th-11th 2016, Crete Island, Greece

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Outline



Industrial Motivation

Numerical Schemes

Test Cases:

Inviscid Vortex Transport DNS of Taylor-Green Vortices at Re = 1,600 LES of a Circular Cylinder at Re =3,900

Conclusions

Industrial Motivation



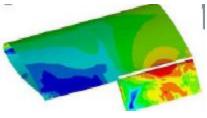
High-Order CFD targeted for high fidelity simulations

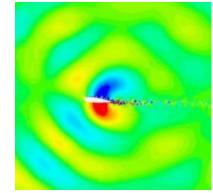
Aerospace

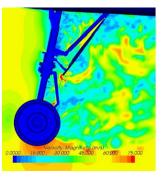
- wing transition,
- high-lift devices
- engine noise
- landing gear aeroacoustics Automobiles/Trucks
- full vehicle aerodynamics,
- mirror, window, sunroof aeroacoustics,
- HVAC fans, ducts, nozzles, turbochargers Combustion
- gas turbine, reciprocating engine
 Nuclear (steam line/T-junctions, etc.)
 Wind turbines

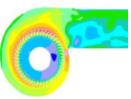
Numerical solution of (unsteady) turbulent flows requires:

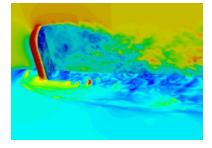
- Accuracy (+ energy conservation, realizability)
- Robustness (large industrial meshes)
- Efficiency (CPU, 10-100 k cores)

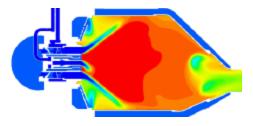












Numerical schemes

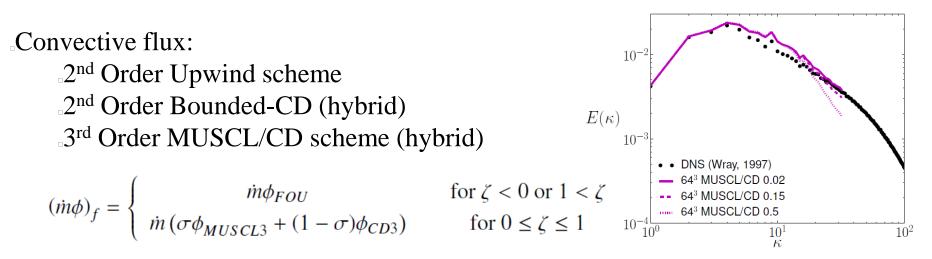


STAR-CCM+, cell-centered FV solver of arbitrary cells Two Solvers:

Coupled approximate-Riemann solver for all flow regimes

Segregated Rhie-Chow SIMPLE or PISO based solver

Least-Squares gradient reconstruction with high-fidelity gradient limiters Preconditioning and other algorithm enhancements for high-fidelity simulations BDF 2 and optimized BDF2 (with reduced numerical error) implemented LES models (WALE, Smagorinsky and dynamic Smagorinsky)



blending based on local smoothness indicator via Normalised Variable Diagram (Darwish et al.)

Inviscid Vortex Transport



"Slow vortex": $M_{\infty} = 0.05, \beta = 1/50, R = 0.005.$ U normalised 1 010 **Solution** Initial T= 50 1.006 Solution **T=0** 1.002 0.9980 0.9940 0.9900 $\delta u = -(U_{\infty}\beta)\frac{y-Y_c}{R}\exp\left(\frac{-r^2}{2}\right)$ $\delta v = (U_{\infty}\beta)\frac{x-X_c}{R}\exp\left(\frac{-r^2}{2}\right)$ $\delta T = \frac{1}{2C_p} (U_\infty \beta)^2 \exp(-r^2)$ $u_0 = U_\infty + \delta u$ $v_0 = \delta v$ $C_p = \frac{\gamma}{\gamma - 1} R_{\text{gas}}$ $r = \frac{\sqrt{(x - X_c)^2 + (y - Y_c)^2}}{R}$ $U_{\infty} = M_{\infty} \sqrt{\gamma R_{\text{pas}} T_{\infty}}$

128² 3rd-O MUSCL/CD 128² 2nd-O upwind 256² 2nd-O upwind 256² 3rd-O MUSCL/CD 512² 3rd-O MUSCL/CD

 $512^2 2^{nd}$ -O upwind

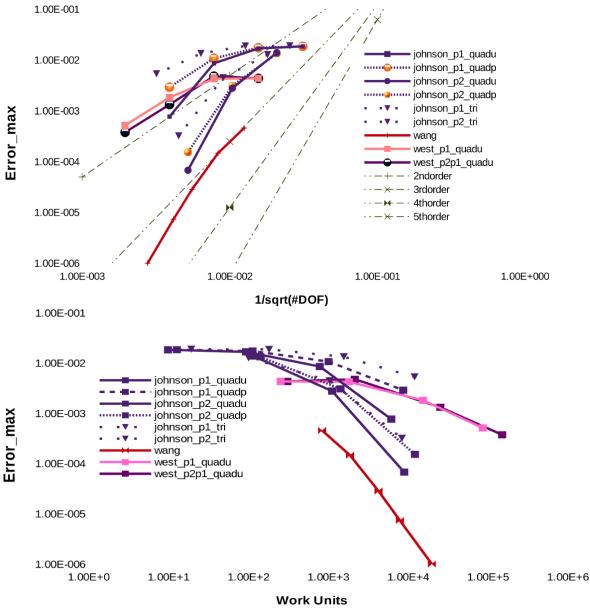
Inviscid Vortex Transport



$$\begin{split} L_2 Error_{|U|} = & \sqrt{\frac{\sum_{i=1}^N \int_{V_i} (|U| - |U|_{initial})^2 dV}{\sum_{i=1}^N \int_{V_i} dV}} \\ |U| = & \sqrt{u^2 + v^2} \end{split}$$

$$WorkUnit = \frac{CPUs \times t_{STAR-CCM+}}{t_{TauBench}}$$

$$h = \frac{1}{\sqrt{nDOF}} = \frac{1}{\sqrt{N}}$$





Evolution of quantities:

$$\begin{split} E_k &= \frac{1}{\rho_0 \,\Omega} \int_{\Omega} \rho \, \frac{\mathbf{v} \cdot \mathbf{v}}{2} \, d\Omega \, . \\ \epsilon &= -\frac{dE_k}{dt} \, . \\ \mathcal{E} &= \frac{1}{\rho_0 \,\Omega} \int_{\Omega} \rho \, \frac{\omega \cdot \omega}{2} \, d\Omega \, . \end{split}$$

 $u = V_0 \sin\left(\frac{x}{L}\right) \cos\left(\frac{y}{L}\right) \cos\left(\frac{z}{L}\right) ,$

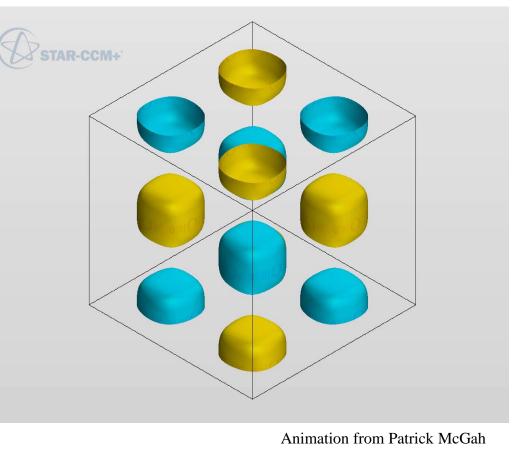
 $v = -V_0 \cos\left(\frac{x}{L}\right) \sin\left(\frac{y}{L}\right) \cos\left(\frac{z}{L}\right) ,$

 $p = p_0 + \frac{\rho_0 V_0^2}{16} \left(\cos\left(\frac{2x}{L}\right) + \cos\left(\frac{2y}{L}\right) \right) \left(\cos\left(\frac{2z}{L}\right) + 2 \right) \,.$

Initial Conditions:

w = 0.

Isosurface of vorticity magnitude coloured by vertical vorticity

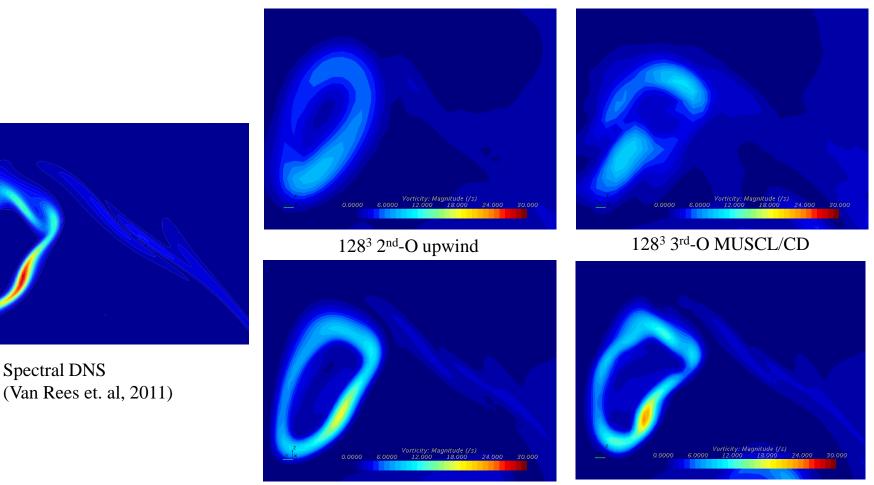


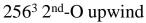


2563 3rd-O MUSCL/CD

Contours of dimensionless vorticity magnitude $|\omega| L/V_0$ at $x = -\pi L$ and $t_c = 8$.

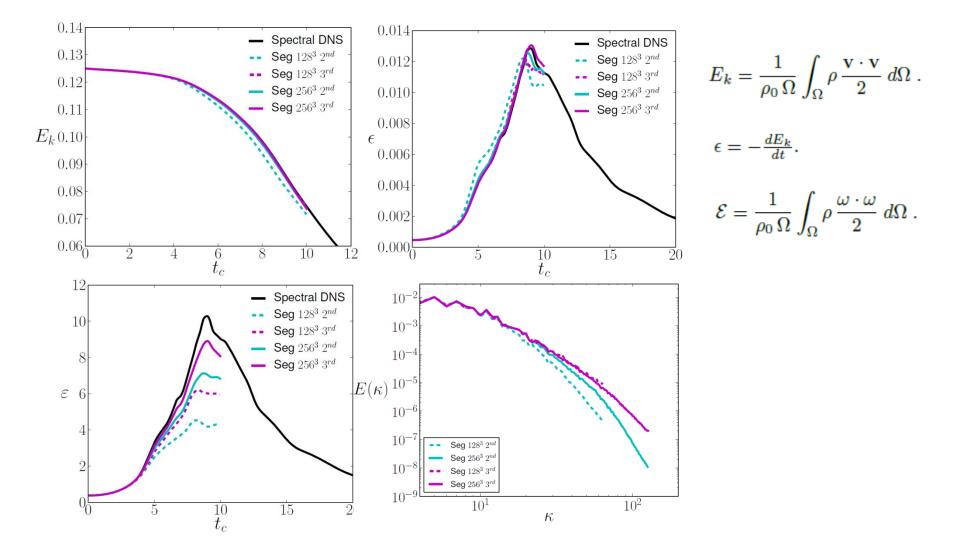
Spectral DNS

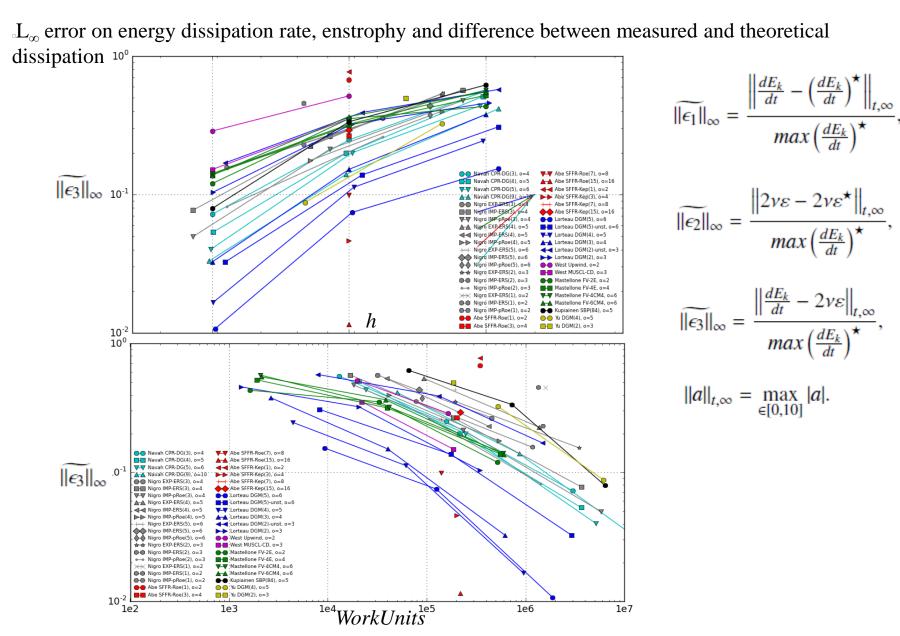






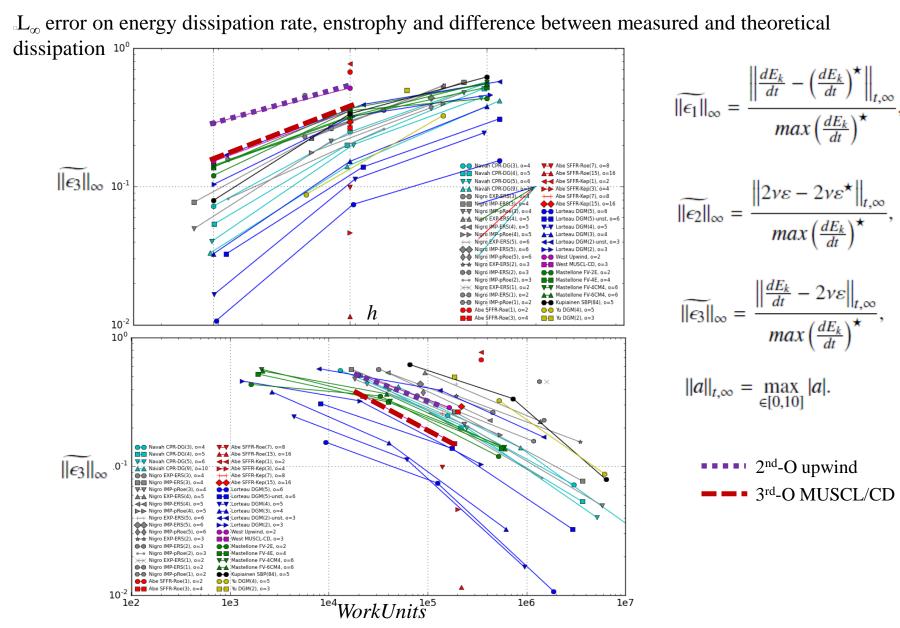
Temporal evolution of total kinetic energy, energy dissipation rate and enstrophy dissipation













Ma = 0.2 x [-9D, 25D], y [-9D, 9D], z [0, π D]

Coupled implicit solver

periodic in z, free-steam in, pressure outlet for other outer boundaries

Initialised with the Grid-sequencing (solves inviscid flow on coarse-grid levels based on the linear-system coefficients, from coarsest mesh to fine mesh)

WALE subgrid-scale model (none for PyFR)

Time statistics collected over 100-1100 t_c

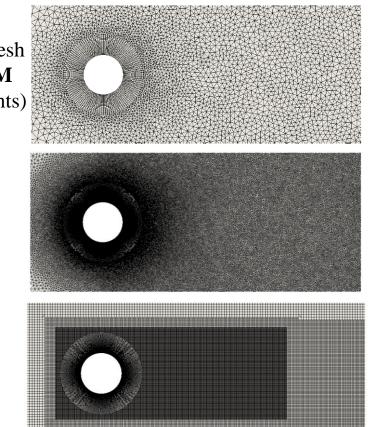
dt = 8.5e-2 tc (3O) dt = 5.0e-3 tc (2O) dt = 2.4e-4 tc (PyFR)

Ran on 256 cores for 2-3 weeks

Prismatic+ Tetra. HO mesh (307K, **13.9M** on P4 elements)

Prismatic+ Tetra. mesh (**13.2M**)

Prismatic+ hexa. mesh (13.5M)



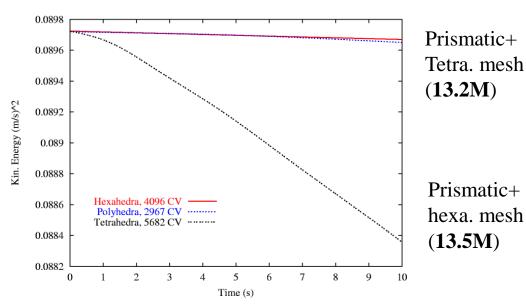


Ma = 0.2x [-9D, 25D], y [-9D, 9D], z [0,πD] Coupled implicit solver

periodic in z, free-steam in, pressure outlet for other outer boundaries

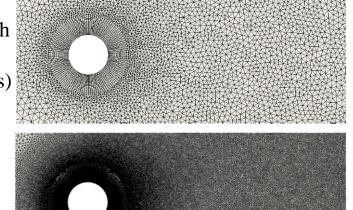
Initialised with the Grid-sequencing (solves inviscid flow on coarse-grid levels based on the linear-system coefficients, from coarsest mesh to fine mesh)

Tetrahedral meshes much more diffusive (see below conservation of kinetic energy for 2D Taylor Green)



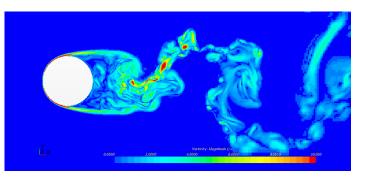
Prismatic+ Tetra. HO mesh (307K, **13.9M**

elements)



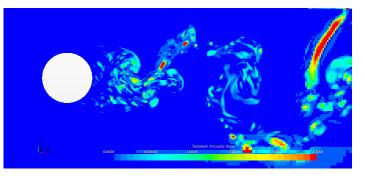


	L _z /D	$t_c U_{\infty}/D$	$\mathbf{f}_{\mathbf{vs}}$	C _d
Present LES (30 MUSCL/CD)	π	1000	0.215	1.053*
DNS (Lehmkuhl et al. 2013)	π	3900	0.211	1.015
DNS (Ma et al. 2000)	2π	480	0.203	0.96
DNS (Tremblay 2002)	π	240	0.22	1.03
LES (Kravchenko & Moin, 2000)	π	35	0.21	1.04
Exp. (Norberg 1988) (Re=3,000)	67		0.22	0.98



* C_d only over 280 t_c

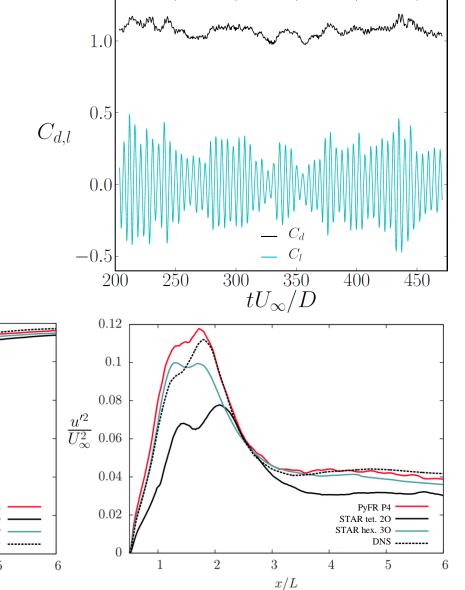
Velocity magnitude, vorticity magnitude & SGS viscosity ratio

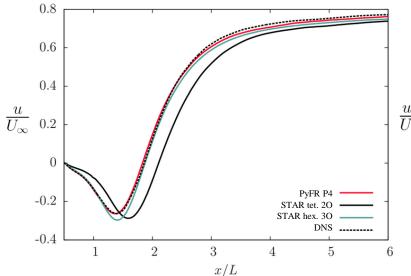




Time evolution of lift and drag showing low-frequency unsteadiness/undulations

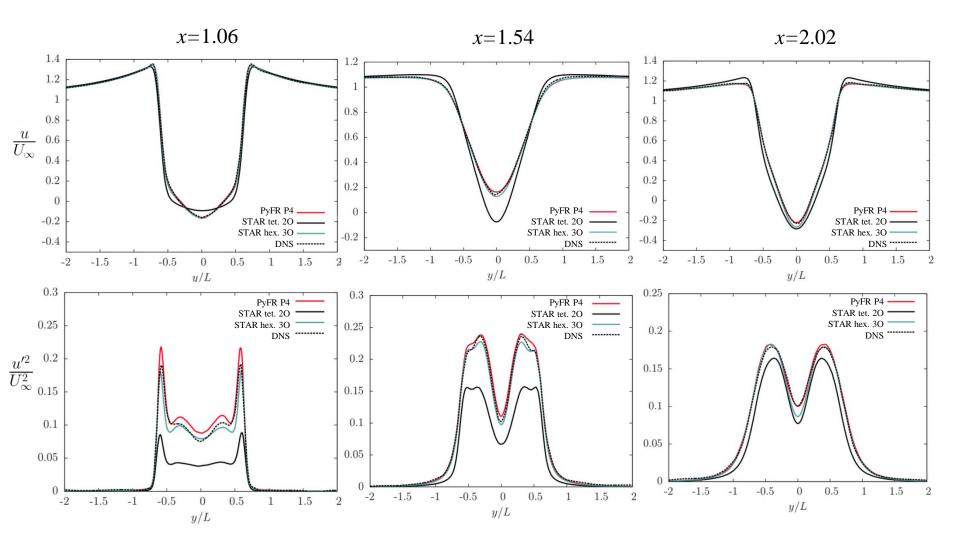
Time and Spanwise averaged streamwise velocity and fluctuations along streamwise direction from the cylinder surface



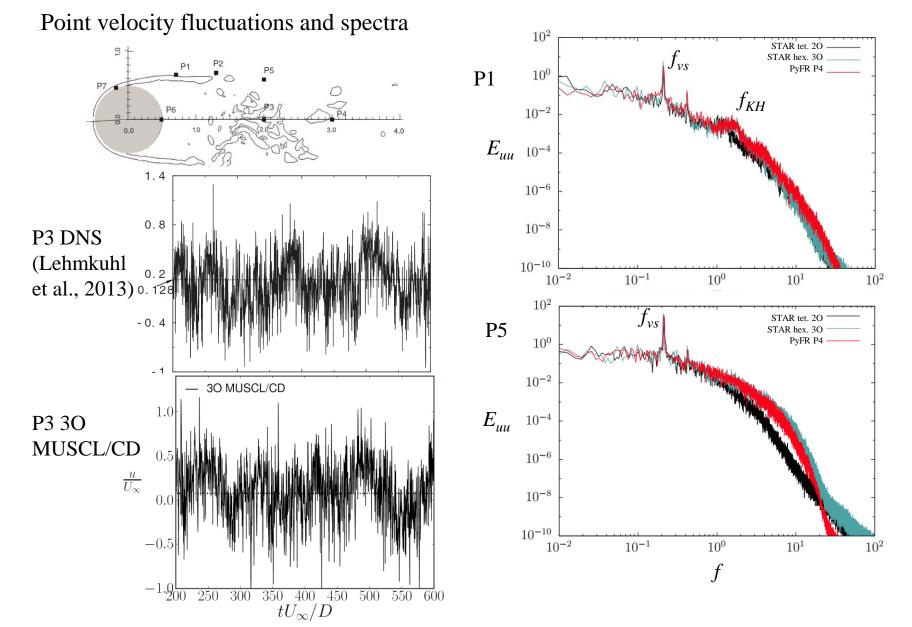




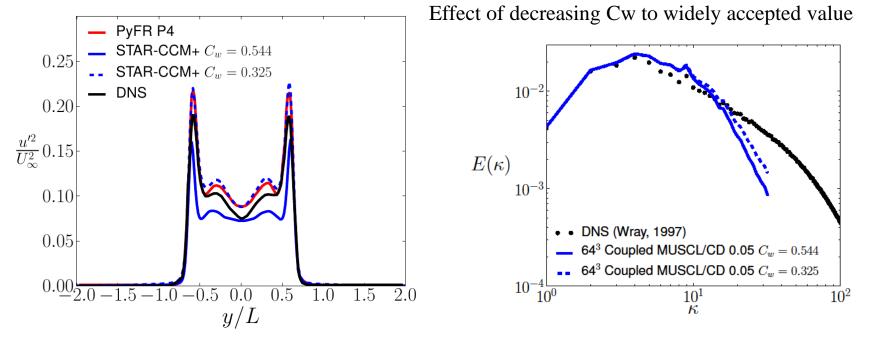
Time and Spanwise averaged streamwise velocity and fluctuations along streamwise slices











Computational resource:

	Hardware	CUDA C Cores/core s x nodes	Price (£)	GPU/ CPU hours	Resource Utilization (£×sec)
PyFR P4	Nvidia K20c GPU	2496x3x12	~2000	4.24×10^{4}	3.05x10 ¹¹
2O upwind	Xeon X5650 CPU	6 x 5	~700	6.96x10 ⁴	1.76x10 ¹¹
30 MUSCL/CD	Xeon X5650 CPU	6 x 5	~700	1.39x10 ⁵	3.51x10 ¹¹

Conclusions



^{3rd} Order hybrid MUSCL/CD scheme was validated on fundamental test cases

Formal order accuracy still to be proven on fundamental cases

^a 3rd Order scheme compares well against higher-order codes for more complex high-fidelity simulations (both in terms of accuracy and compute resource)

Work still needed to single out contibution to mesh and convection scheme

Future work on proving formal convergence rate (using other HO workshop cases)

Acknowledgements:

We would like to thank B. Vermeire and P. Vincent at Imperial College for providing both 'naive-user' STAR-CCM+ settings and PyFR P4 solution for the ciruclar cylinder.